

Laboratory Test Facilities example: Heat Pump Water Heaters for Demand Response Policies

Abstract — The development of Demand Response in residential segments is basic to develop a practical flexibility of demand, because these segments account for up to 40% of the overall demand. Energy Efficiency is another concern for these segments, but unfortunately present scenarios lack a practical coordination between Efficiency and Demand Response. This paper deals with an important problem in residential Demand Response: the determination of the flexibility and response on the demand-side, in this case through loads which can have a high potential for Demand Response and also a considerable interest for energy savings: Heat Pump Water Heaters. A residential load has been fully monitored (temperature, consumption, water flow) in the laboratory to obtain a Physically-Based Model which allows the evaluation of Demand Response options. Moreover, the model helps the aggregator obtain how the flexibility of demand (power, energy, energy payback or rebound effects) can be modified or limited, and how to deal with these characteristics and limitations to engage customers in Electricity Markets.

Keywords—Demand Response, Renewable Sources, Energy Markets, Load Modeling, Energy Storage, Energy Efficiency

I. INTRODUCTION

The increasing participation of Renewable Energy Sources (RES) in the generation mix withdraws a considerable amount of flexibility in the Supply-Side. This lack of flexibility can be balanced in the Demand Side through the use of Demand Response (DR) and Energy Storage [1]. This flexibility is also of interest for designing the new internal market of electricity in the EU [2], in where the customer should play a new and more active role through energy aggregators, but these tasks need the development of new tools and methodologies such as that proposed in this work.

The main end-uses in residential segments according to energy consumption reports in Spain are: Electrical Heating (42.9%), Cooking (7.69%), Lighting (4.85%), Water Heaters (17.96%), Air Conditioning (0.98%) and other appliances (25.5%). This picture of end-uses is similar for other industrialized countries [3]. In this way, Electric Heating (EH) and Water Heating (WH) are first candidates to participate in Demand Response policies. Moreover, WH has an inherent capacity for energy storage and this is interesting for the response of residential customers to dynamic price tariffs, or also to consider thermal storage as an alternative to electrical storage (for example, for customers that own some kind of generation, the so called “prosumers”).

The idea of this paper is to develop and validate a load model (Physically-Based, PBLM) which is able to take profit from the possibilities of a “new” load: the Heat Pump Water Heater (HPWH) which also has a remarkable interest from the point of view of Energy Efficiency. From an economic point of view HPWH can help customer to reduce energy costs, or in other cases this load allows the participation of customers in Capacity Markets (through an aggregator) in both Energy Efficiency and Demand Response options [1]. The paper is organized in different sections. In Section II, the characteristics of HW loads are discussed; the PBLM model is presented, and validated through some tests. Then, in Section III, some DR simulation examples are described to show the ability of the model, followed by Section IV that concludes the paper.

II. METHODOLOGY

Several models have been proposed in the literature for the inclusion of WH in the Demand Response portfolio [4-7]. The approach presented in [4] presents the idea of one or two node WH models which changes the height of the hot water (and model) to take into account the stratification of water, whereas [5] solves the problem of water stratification in the reservoir through Dynamic Fluid equations. This makes much more complex the model but it gains in accuracy. Other approaches use well know software platforms (EnergyPlus) to solve heat transfer processes in HPWH [6]. In [7] a learning-based, data-driven model is developed by using a nonlinear autoregressive network with external input. Mains problem of these approaches are that usually consider conventional WHs, and some of them [6] do not search for a physical explanation of load behaviour. Note that Heat Pump alternative is very sound from the point of view foreseeable participation of residential segments in capacity markets [8].

Fig. 1 shows two tests during the start of two WH (with different technologies). As it can be seen, the start and standby periods are different, but mainly they differ when the appliance is switched on and the reservoir is empty: i.e. charging time and standby cycling (and this is of interest in the case of the application of DR policies on these loads). This different behaviour and the importance of this end-use in residential segments justify the monitoring of this kind of load.

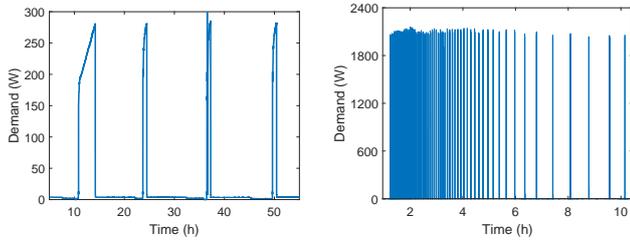


Fig.1. Water Heater behaviour: a) Heat Pump Water Heater (AristonNuos Evo 80l); b) Resistance (conventional) Water Heater (Junkers 75l)

A. Laboratory test facilities

The HPWH load that is considered in this work has two different heat sources: an auxiliary resistor and the Heat Pump, with three modes of service (“eco”, “boost” and “auto”, which select the use of compressor, resistor or both sources of heat according to customer choices and boundary conditions of the load and its environment). This duality in supply presents problems (rebound effects, peak demand due to HP inefficiencies at low temperatures in mode auto) and opportunities (more flexibility for DR if the customer or aggregator can change remotely the options for main supply of energy). Both are important concerns about what the paper aims to discuss. To evaluate the dynamic of the HPWH, a load has been installed in the Area of Electrical Engineering of the Universidad Miguel Hernández (Spain). The characteristics of load are shown in table I.

According to the experience described in [5], a broad monitoring system has been installed on the load and in its environment. Fig. 2 shows a scheme of the laboratory test facilities.

TABLE I. CHARACTERISTICS OF HPWH ARISTON NUOS EVO

Characteristic	Value
Capacity (l), Energy Label	80l, A+
Rated Power of Heat Pump (W)	250 (avg)/ 350 (max)
Performance, COP (outdoor air at 7°C)	2.55
Max. Heating time (h)	5h35m
Max. WH Temperature (°C)	55 (only HP)/ 62
Auxiliary Resistor (W)	1200

This system measures outdoor temperature and humidity, temperature of water flows and pipelines (especially cold and hot temperatures during water draws), the electrical consumption of the auxiliary resistor and the compressor of heat pump by means two independent electrical meters, one of them a switch with control of supply to simulate several DR policies. The system also has a water flow controller to simulate several water draw profiles. All the sensors and controllers use the Z-Wave protocol, except the secondary water flow meter which sends pulses to a data logger (1000 pulses/l). Load monitoring and power management is done by the automation software platform IP-Symcon [9] because this platform allows the management of several protocols at the same time (1-wire, F10, KNX, EnOcean, M-BUS, Modbus, Oregon Scientific, Siemens OZW/S5/S7 and Zwave).

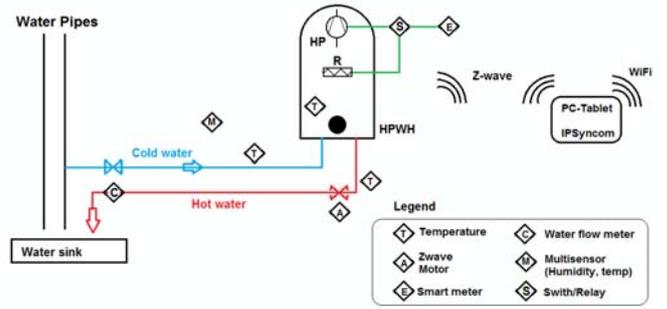


Fig.2. Scheme for measurement and control system for HPWH load

Fig. 3 shows some pictures of the load, sensors and actuators. The system has five sensors to measure: inlet and outlet water temperature, water flow, humidity and internal temperature of laboratory. Moreover, an actuator to control water flow, two demand meters and a controlled plug have been deployed. The interval between measures can be selected by php scripts from 5s to hours. In our case Zwave devices and IPSymcon are programmed to notify and record any change in variables (>5%).

B. Elemental model of HPWH

The PBLM model proposed for HPWH is a thermal-electric equivalent, specifically a lumped $RxCx$ network being fed by two heat (current) sources. The model takes into account the water storage capacity in the tank (C_x parameter), heat losses (G_x), the heat losses due to inlets of cold water (a dependent current source in the model), outlets of hot water (again a dependent current source) and heat gains due to auxiliary resistor (H_R) or Heat Pump (H_{HP}) both current sources. The resistor and HP sources are selected by the user through the appliance menu of the load (MODE eco or boost).

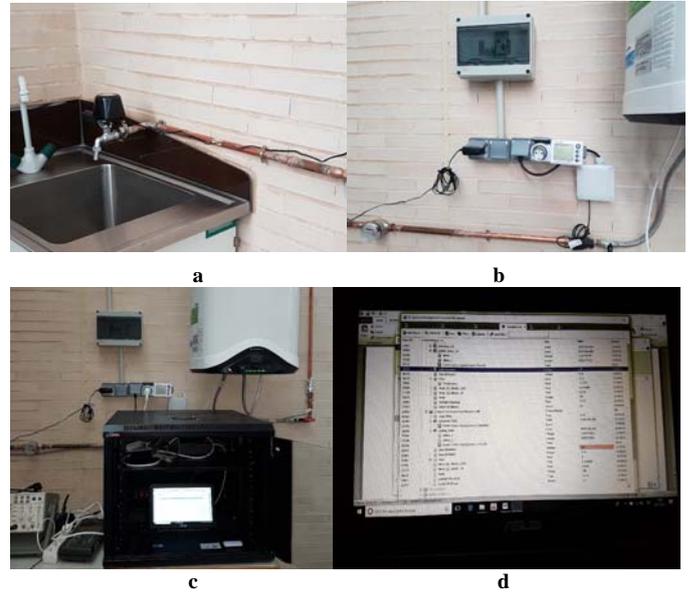


Fig. 3. HPWH monitoring: a) Valve for water flow control; b) Zwave Switch/meter (Fibaro), flow meter (Sensus ResidiaJet), temperature sensors (Qubino and DS18B20) and energy logger (Volcraft); c) Overall system and rack for sensor supply sources, Zwave gateway and pulse transduction of secondary flow meter; d) PC and IPSymcon recording Zwave devices.

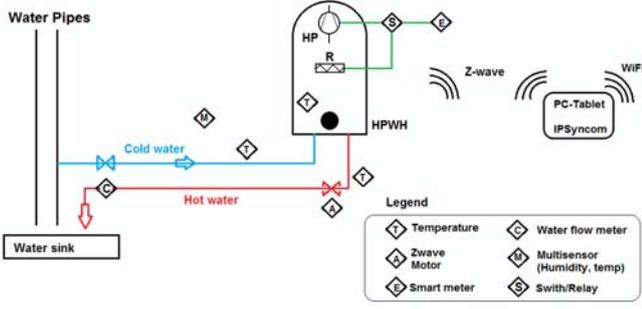


Fig. 4. The electrical-thermal equivalent of a HPWH load.

The state-space representation for the model in fig. 4 is:

$$\begin{pmatrix} DX_1(t) \\ DX_2(t) \end{pmatrix} = \begin{bmatrix} \frac{G_L + G_{C1} + G_{E1} + c_e q(t)}{C_1} & \frac{1}{C_1} G_L \\ \frac{1}{C_2} G_L & -\frac{G_L + G_{C2} + G_{E2}}{C_2} \end{bmatrix} \begin{pmatrix} X_1(t) \\ X_2(t) \end{pmatrix} + \begin{bmatrix} \frac{1}{C_1} G_{e1} & 0 & \frac{1}{C_1} \\ \frac{1}{C_2} G_{e2} & \frac{c_e q(t)}{C_2} & 0 \end{bmatrix} \begin{pmatrix} X_d(t) \\ X_p(t) \\ H_{HP}(t) \end{pmatrix} \quad (1)$$

Where:

- $m(t)$: is the control mechanism which drives the demand: a thermostat in thermostatically controlled loads.
- X_1 and X_2 : are the state variables that are temperatures; the temperature of the water inside the HPWH.
- $q(t)$: water flow, i.e. the service of the load, in this case flow at a certain temperature level X_s (thermostat setpoint). This variable explains the energy requirements through specific heat c_e of water and inlet/outlet temperatures.
- MODE: the switch that drives the mode of use of the load (with an auxiliary resistor or the compressor).

The capacity of the reservoir (water tank) in the proposed model is split in two blocks (WH-1 and WH-2). The reason is that water heater suffers the so called water stratification (the hot water raises to the top of the tank reservoir and the cold water down to the bottom). This phenomenon has been described in the literature [6] and analysed through Fluid Dynamic equations. The model is developed using building energy simulation programs to model energy consumption, for example EnergyPlus platform [10]. In this paper, a more simple “grey-box” model is proposed which allows the interoperability with other platforms such as BRCM toolbox [11] and the possibility to apply aggregation methodologies previously applied for other PBLM models (HVAC) developed by authors [12].

A detailed analysis of ON times, but especially of OFF times in the load tests (as shown in fig. 1), reflects that a “mixed tank model” (i.e. the premise that a homogeneous temperature in the tank exists) does not work very well. Tests show that ON times are nearly constant (fig 5.a) whereas OFF times go up during a certain time but then they remain constant (fig 5.b). The switching pattern shows the changes in charging/storage processes due to hot water convection flows

(i.e. the water near the condenser coil of HP or the resistor is heated, but the flow of warm/cold water mixes flows and drops the internal temperature; then the thermostat goes ON again). When this mixing process, driven by Fluid Dynamic laws, in the tank finishes, switching times are only due to thermal losses from tank to the external dwelling. For these reasons, a two-mixed tank model is preferred to reflect the stratification of the tank shown in tests (fig 5b).

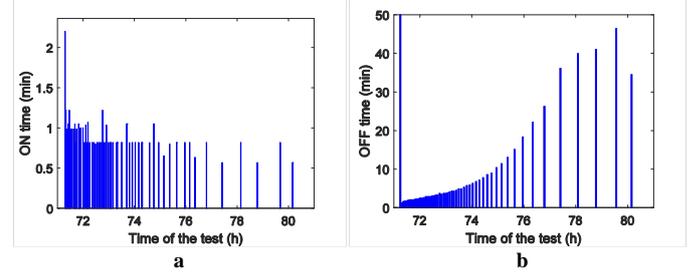


Fig. 5. ON and OFF times during the real test shown in figure 1 (conventional WH or HPWH in mode resistor): a) ON times; b) OFF times.